

Claims

1 1. Dual clutch arrangement comprising a torsional vibration damper arrangement (12)
2 with a primary side (16) which can be fixedly coupled with a driving member for joint rotation
3 about an axis of rotation (A) and a secondary side (24) which is rotatable about the axis of
4 rotation (A) against the action of a damper element arrangement (37) with respect to the primary
5 side (16), and a dual clutch (14) with an input area (90) and two output areas (66, 76), each of the
6 output areas (66, 76) being coupleable with one of two driven members so as to be fixed with
7 respect to rotation relative thereto, wherein the secondary side (24) of the torsional vibration
8 damper arrangement (12) is supported with respect to the primary side (16) of the torsional
9 vibration damper arrangement (12) in axial direction and/or in radial direction by the input area
10 (90) of the dual clutch (14) and a bearing arrangement (62, 88) which supports this input area
11 (90) with respect to a stationary subassembly (80).

1 2. Dual clutch arrangement according to claim 1, characterized in that the stationary
2 subassembly (80) comprises a transmission housing (80).

1 3. Dual clutch arrangement comprising a torsional vibration damper arrangement
2 (12a) with a primary side (16a) which can be fixedly coupled with a driving member for joint
3 rotation about an axis of rotation (A) and a secondary side (24a) which is rotatable about the axis
4 of rotation (A) against the action of a damper element arrangement (37a) with respect to the
5 primary side (16a), and a dual clutch (14a) with an input area (90a) and two output areas (66a,
6 76a), each of the output areas (66a, 76a) being coupleable with one of two driven members so as

7 to be fixed with respect to rotation relative thereto, wherein the input area (90a) of the dual
8 clutch (14a) is supported with respect to the secondary side (24a) of the torsional vibration
9 damper arrangement (12a) on the one hand and with respect to a stationary subassembly (80a)
10 on the other hand by a flexible coupling arrangement (98a, 104a).

1 4. Dual clutch arrangement according to claim 3, characterized in that the stationary
2 subassembly (80a) comprises a transmission housing (80a).

1 5. Dual clutch arrangement according to claim 3 or 4, characterized in that the flexible
2 coupling arrangements (98a, 104a) permit a movement of the input area (90a) of the dual clutch
3 (14a) in radial direction and/or in axial direction.

1 6. Dual clutch arrangement according to one of claims 3 to 5, characterized in that the
2 secondary side (24a) of the torsional vibration damper arrangement (12a) is supported by an
3 axial bearing (94a) and a radial bearing (96a) at the primary side (16a) of the torsional vibration
4 damper arrangement (12a).

1 7. Dual clutch arrangement comprising a torsional vibration damper arrangement
2 (12b) with a primary side (16b) which can be fixedly coupled with a driving member for joint
3 rotation about an axis of rotation (A) and a secondary side (24b) which is rotatable about the axis
4 of rotation (A) against the action of a damper element arrangement (37b) with respect to the
5 primary side (16b), and a dual clutch (14a) with an input area (90b) and two output areas (66b,
6 76b), each of the output areas (66b, 76b) being coupleable with one of two driven members so as
7 to be fixed with respect to rotation relative thereto, wherein the secondary side (24b) of the

8 torsional vibration damper arrangement (12b) is supported with respect to the primary side (16b)
9 of the torsional vibration damper arrangement (12b) by an axial/radial bearing arrangement
10 (110b) which permits tilting of the secondary side (24b) with respect to the primary side (16b).

1 8. Dual clutch arrangement according to claim 7, characterized in that the secondary
2 side (24b) of the torsional vibration damper arrangement (12b) is coupled with the input area
3 (90b) of the dual clutch (14b) by a coupling arrangement (134b) so as to be fixed with respect to
4 rotation relative to it, which coupling arrangement (134b) permits a tilting of the secondary side
5 (24b) with respect to the input area (90b).